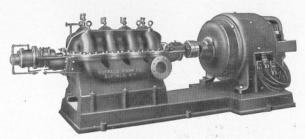
BUFFALO

Double Suction Impeller Multistage
Centrifugal Pumps



CLASS "RDS"

Bulletin No. 952 A

Buffalo Steam Pump Co. Buffalo, N. Y.

New York Boston Philadelphia Pittsburgh Cleveland Detroit Chicago St. Louis Los Angeles New Orleans Atlanta Minneapolis Denver

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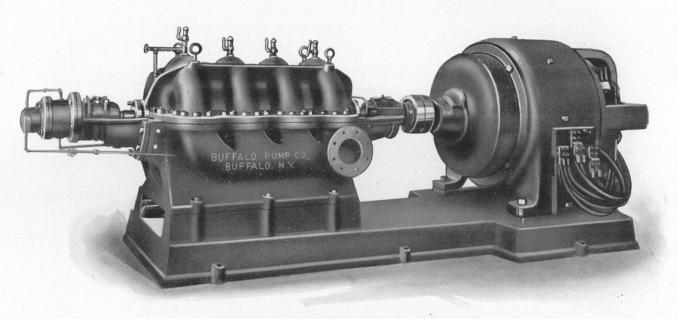


Fig. 1247
4 Stage Class "RDS"--Motor Driven

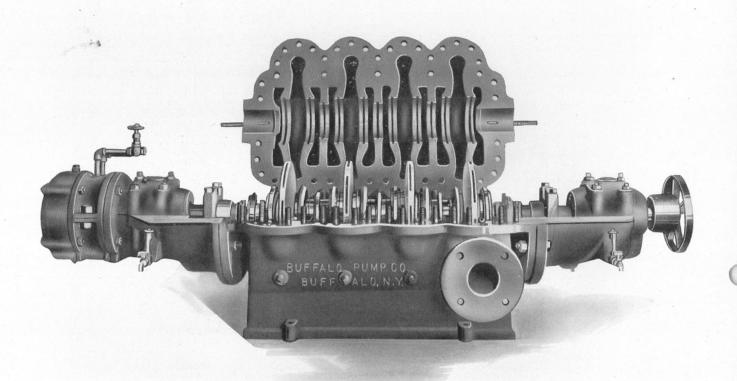


Fig. 1245
4 Stage Class "RDS"---Cover Raised

Buffalo Class "RDS" Multi-Stage Centrifugal Pumps

B UFFALO" Class "RDS" Multi-stage double suction impeller centrifugal pumps are a special development of all the best features in pump design, selected during our long experience as pump manufacturers, and are designed and built to meet the requirements of the most exacting service.

These pumps are being used by large manufacturing concerns, public service corporations, city water works, and the United States Government. We particularly commend their investigation by discriminating engineers and users requiring pumps for Public Water Supply, Municipal and Private Fire Service, Boiler Feeding, Mill and Factory Water Supply, or any other service where thorough reliability and high efficiency are important considerations.

The "Buffalo" Class "RDS" pump is of the horizontal shaft "double suction" impeller multistage type, with horizontally divided casing—and is built in two, three or four stages, to suit prescribed conditions.

These pumps are offered for capacities up to 2000 U. S. Gallons per minute. The standard pumps are built for 250 lbs. per square inch maximum working pressure and all standard pumps are strong enough to withstand this pressure with a large factor of safety. Special extra heavy pumps can also be furnished for maximum working pressures up to 400 lbs. per square inch.

The Class "RDS" pumps are especially designed for direct connection to standard high speed and moderate speed electric motors and steam turbines. They are also adaptable for pulley drive.

CASING:

The casing is the horizontally divided type, the suction and discharge nozzles being cast in the lower half, permitting the removal of the upper half so the interior parts can be inspected or removed without disturbing the pipe connections.

The suction and discharge nozzles are located on opposite sides of the pump casing and pumps may be arranged for either right or left hand rotation. The casing has machined joints, and flanges are secured together with heavy bolts.

The construction of the water passages throughout is of the "double suction volute" type, and because of this special construction the pump, when in operation, is in perfect hydraulic balance under all conditions of head and pressure. The water passages are simply formed and of ample areas to avoid friction losses and sudden changes of velocity.

The water enters each stage through a "double suction volute" chamber, formed to maintain a practically constant velocity at all points of the impeller inlets. After passing through the impeller the water is delivered into a discharge volute which is also formed to maintain constant velocity of flow and from which the water is discharged into a pressure chamber, which forms the water channel leading to the next impeller. This pressure chamber is formed so that the velocity of water passing through it is gradually decreased up to the center of the passage and from there is again increased to the proper velocity for entrance to the next impeller. This permits of conversion of velocity into pressure, between each two stages, with the least possible loss due to shock and friction. Another important feature of our suction and discharge volute design is that the direction of flow or "whirl" is not changed from the time the water enters the first suction volute until it is discharged from the discharge volute of the last stage. This is a valuable improvement in that it eliminates the energy required to change the direction of flow of a column of water moving at high velocity.

IMPELLERS:

The impellers are the enclosed "double suction" hydraulically balanced type, the water entering the impellers at opposite sides in equal volume and pressure; thereby securing the perfect

hydraulic balance previously referred to. Each impeller individually and the entire series of impellers as a whole are therefore in hydraulic balance under all conditions of operation.

While this construction ensures perfect hydraulic balance under operating conditions we equip each pump with a marine type thrust bearing. This maintains the impellers in a central position in the casing when starting the pump, and takes up thrust if one impeller becomes obstructed with foreign matter.

The impellers of standard pumps are made of hard, close grained cast iron. For boiler feeding where hot water is handled and for some other special services, we recommend that pumps be fitted with brass or bronze impellers.

The impellers are very carefully machined and hand dressed to ensure smooth surfaces and perfect rotary balance. Each impeller is tested separately for rotary balance, and after mounting, the entire rotor, including impellers, shaft, keys and coupling is placed in perfect rotary balance. The shaft is enlarged in diameter where the impellers are mounted on it.

SHAFT:

The shaft is made of high grade open hearth steel machined all over, and the impellers are mounted on an enlargement of diameter at about the center of the shaft and secured by means of steel feather keys. Each impeller is secured by its individual key and the keyways are spaced radially around the shaft so as to maintain its strength to the fullest extent.

The shaft has a shoulder turned in place to locate the first impeller mounted thereon; between each pair of impeller hubs the shaft is protected by a bronze sleeve with flanged ends which acts as a guide for the water entering the impellers and also completely covers the ends of the feather keys. At each end of the impeller equipment there is provided a bronze covering nut, threaded on the shaft, which permits of a slight adjustment endwise in assembling the pump and also forms a protecting cover for the outer ends of feather keys and the joints between the shaft and the outer shaft sleeves. These sleeves are made of bronze tubing and are extended from under the impeller nuts to beyond the stuffing box glands at each side of the pump. Sleeves are forced on the shaft under hydraulic pressure.

CLEARANCE RINGS:

The clearance rings are the flat surface floating type, made of bronze and carefully machined all over. These rings are located around the suction openings of each impeller and surrounding

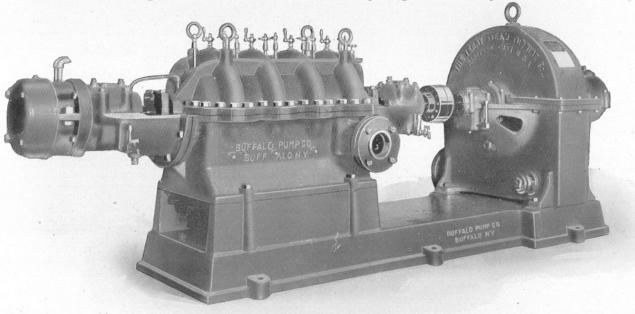


Fig. 1246 4 Stage Class "RDS".--Steam Turbine Driven

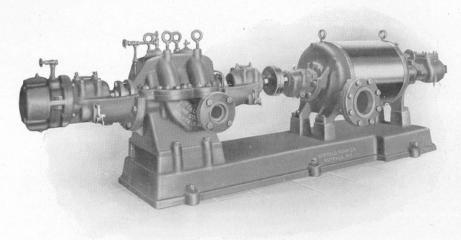


Fig. 1248 2 Stage Class "RDS"—Steam Turbine Driven

the shaft between each two stages. They are the "L" section type ensuring strength without undue weight, and are arranged so that they are held in position by the difference in pressure between the stages, thus ensuring a tight joint between the stages, and this also prevents them from turning with the impellers. This arrangement eliminates friction and leakage.

SHAFT BEARINGS:

Pump bearings are located at each end of the casing and consist of

heavy cast iron bracket housings with flanges bolted to the pump casing and centered in turned and bored fittings.

The bracket forms a drip pocket and support for the horizontally split bearing shell which consists of a cast iron jacket, turned on the outside and fitted to a bored seat in the bracket. The bearing shells are horizontally split and lined with high grade babbitt metal of ample thickness, peined, bored and scraped to a perfect fit with the shaft.

The bracket is provided with an ample oil reservoir and the bearings are lubricated by means of brass oil rings, suitable provision being made for returning oil to the reservoir after passing through the bearing; also for filling and draining. Each bearing is provided with a brass cased sight oil gauge.

THRUST BEARING:

Thrust bearing is the multiple collar marine type, and consists of a horizontally split and babbitted housing with flange, bolted to the outer end of outer pump bearing. The thrust collars are machine steel turned from a solid forging, bored to fit over the outer end of pump shaft, and

and nut. The thrust bearing is provided with an oil chamber connected by a channel with the outer pump bearing, and a copious supply of oil is circulated over the thrust collars at all times by means of a small brass oil wheel revolving in the oil chamber and feeding the oil through suitable circulating passages.

The housing and end thrust cover are provided with water jacket arranged to permit a proper circulation of cooling water to maintain the oil at a proper temperature for lubrication.

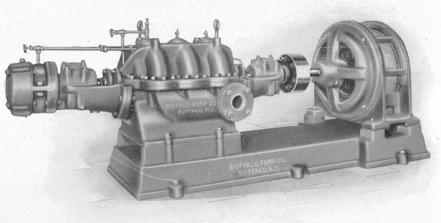


Fig. 1249
3 Stage Class "RDS"---Motor Driven

GLANDS:

Stuffing boxes are cast in each end of the pump casing. They are deep and provide for an ample amount of packing. The throats are fitted with brass bushings.

Packing glands are cast iron with brass bushings and circular flange secured to the stuffing box flange with three large studs and adjusting nuts.

WATER SEALS:

Water seals are provided at each stuffing box, these consisting of brass cage rings with suitable circulating holes. On the suction end, the water seal is provided with water supply connection to prevent air from entering the pump. On the discharge end the water seal is provided with a small release pipe which relieves the discharge pressure from the outer rings of packing which do not require forcing up to prevent leakage. Thereby a cool running packing is provided.

COUPLING:

Shaft coupling is the flexible type. On the small sizes, 2'' and $2\frac{1}{2}i''$, the flanges are connected together with a sole leather plate, having heavy leather segments securely riveted to the plate, and which fit into corresponding openings in the coupling flanges. All other sizes are equipped with couplings of the steel pin and rubber bushing type. In all cases the couplings are of ample strength to carry maximum loads required and are carefully machined and in rotary balance.

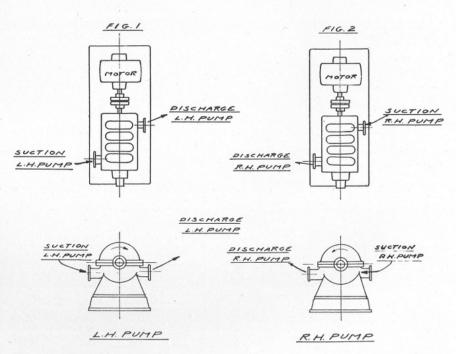
SUB-BASE:

Sub-base is cast iron of very substantial construction, suitably ribbed, and provided with heavy lugs cast on the outside for foundation bolts.

Ample machine finished pads are provided on the top of sub-base for mounting and bolting the pump and motor or turbine. In case of pulley driven pumps the sub-base is as just de-

scribed, except it is made with machined pads for mounting pedestal bearings. Pulley driven pumps of the smaller sizes are furnished with the pump shaft extended through the driving pulley and one outboard ring-oiling pedestal bearing. The larger size pumps are provided with a flexible coupling and the driving pulley mounted on a separate length of open hearth steel shaft carried in two outboard ring oiling pedestal bearings. The pedestal bearings are of heavy cast iron construction, the bearing and lubricating equipment being similar to that used in the main pump bearings.

Pulleys are heavy cast iron in one piece, carefully machined and balanced and mounted on shaft with feather key.



Line Drawing Showing Right Hand (RH) and Left Hand (LH) Class "RDS" Pumps.

BUFFALO CLASS "RDS" DOUBLE SUCTION MULTISTAGE CENTRIFUGAL PUMPS

SPECIFICATIONS

CASING: Cast iron, divided on horizontal center line. Suction and discharge openings cast in lower half, allowing interior parts to be inspected or removed without breaking pipe connections.

IMPELLERS: Cast iron, bronze on special order. Enclosed double suction hydraulically balanced type. Each impeller and entire series are in perfect hydraulic balance when pump is in operation. Mounted on shaft by feather keys and lock nuts.

CLEARANCE RINGS: Bronze "L" section, floating type. Located around suction openings of each impeller, and surrounding the shaft between each two stages. Will not turn with impellers. Prevent leakage and eliminate friction.

SHAFT BEARINGS: Horizontally divided shell type. Shells are horizontally split and lined with high grade babbitt, peined, bored and scraped. Ring oiling. Bearings are located at each end of pump casing.

THRUST BEARING: Multiple collar marine type. Horizontally split housing, babbitted. Thrust collars machine steel turned from a solid forging. Housing water jacketed. Oil circulated continuously over thrust collars.

GLANDS: 'Cast iron, brass bushed.

WATER SEALS: Brass cage rings with suitable circulating holes. Provided at each stuffing box. Seal on suction end has water supply connection.

SHAFT: Open hearth steel, machined all over. Enlarged at center where impellers are mounted, and secured by keys. Brass covered where exposed in pump and glands.

SUB-BASE: Cast iron, ribbed and stiffened. Heavy lugs cast on outside for foundation bolts. Pads cast on top for motor or turbine feet or for pedestal bearings if pulley driven.

COUPLING: Flexible type.

PULLEY: Cast iron, one piece. Pedestal bearings cast iron, with bearings similar to main pump bearings.

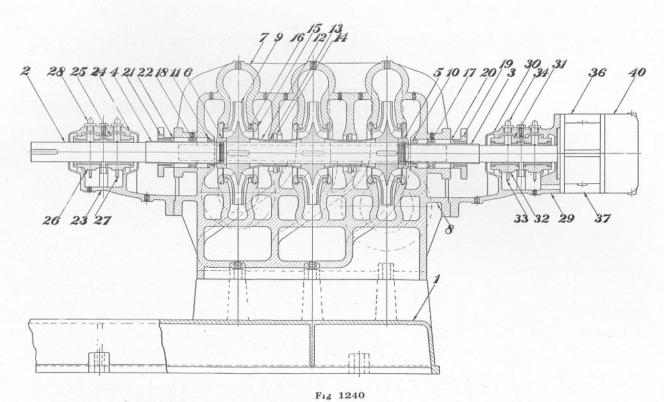
FITTINGS: Drain and air cocks. Piping for water jacket on thrust bearing and for water seal.

FINISH: Painted, filled and rubbed down outside with final finishing coat. Bright parts exposed to weather protected by a slushing compound.

Code Word Standard Pulley	Figure Number	Size of Pump,	Pipe Sizes, Inches		Capacity, Gallons per Minute		Size Pulley, Inches		Approximate Floor Space,	
Driven Pump	rumoci	Inches	Suction	Discharge	Normal	Maximum	Diam-	Face	Inches	
			. 7	WO STAC	E			-		
RSADM RSBER RSCUB RSDIL	1248 1248 1248 1248	2 2 2 3 4	2½ 3 4 5	2 2½ 3 4	100 150 225 400	140 225 325 550	6 7 8 10	5 6 8 10	87x25 87x25 92x27 111x31	
RSEFX RSFAD RSFYK RSFZP	EFX 1248 5A FAD 1248 5B FYK 1248 6		6 5 6 5 8 6 10 8		620 750 620 850 900 1300 1600 2000		10 10 12 13	12 12 16 16	111x31 116x38 120x38 130x43	
			T	HREE STA	G E					
RSGND RSHFZ RSJEQ RSKAP RSLOX RSLVQ RSMIT RSMUX	1249 1249 1249 1249 1249 1249 1249 1249	2 2 3 4 5A 5B 6 8	2½ 3 4 5 6 6 8	2 21/2 3 4 5 5 6 8	100 150 225 400 620 620 900 1600	140 225 325 550 750 850 1300 2000	On Application	On Application	94x25 94x25 99x27 120x31 120x31 127x38 131x38 143x43	
		3 3 3 5 5 6	F	OUR STAC	3E					
RSNUC RSOBY RSPAL RSQEF RSTIZ RSTWD RSWOL RSWOL	1250 1250 1250 1250 1250 1250 1250 1250	2 2½ 3 4 5A 5B 6	2½ 3 4 5 6 6 8	2 21/2 3 4 5 5 6 8	100 150 225 400 620 620 900 1600	140 225 325 550 750 850 1300 2000	On Application	On Application	100x25 100x25 106x27 129x31 129x31 139x38 142x38 155x43	

Add Code Word JCESF for Brass Runners. Add Code Word JCHBY for Brass Glands. Add Code Word JCXRS For Motor Base and Flexible Coupling.

NOTE: Brass Runners are recommended for 2 in. and $2\frac{1}{2}$ in. Pumps, and are necessary on these sizes when speed is 2200 R. P. M. or over.

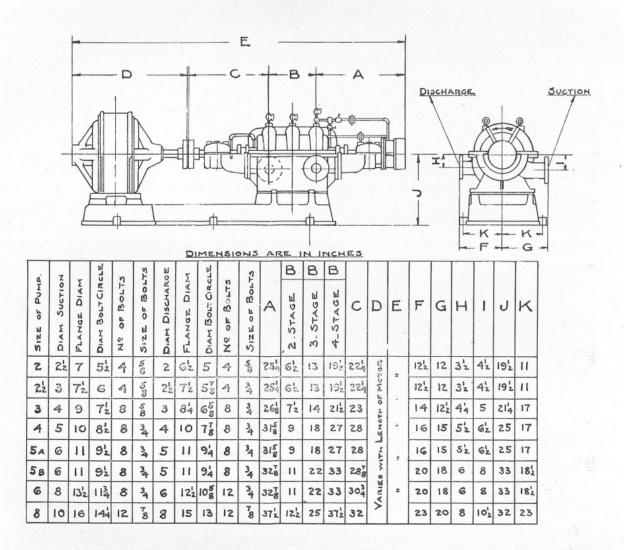


3 Stage Class "RDS"--Sectional View

1	Vo.	Name	Remarks	No.	Name	Remarks	No.	Name	Remarks
		Sub-base			Bearing Bracket		36	Thrust Housing	I'nner [Sometimes]
		Shaft		24.	" Cap	"	37.	" "	Lower piece.
		Shaft Sleeve	Outer	25.	" Bushing	Upper	38.		
	4.	" "	Outer	26.	" "	Lower	39.	" "	Upper Not furnished with two piece hous'g.
	5.	Impeller Nut	Inner	27.	Oil Ring Fo	or Inner Bearing			LOWEL Linece nous g. 3
	6.	" "	Inner	28.	" Cover "	" "	40.	Cover	
	7.	Casing	Top Half	29.	Bearing Bracket	Outer	41.	" Collar	
	8.	"	Bottob Half	30.	" Cap		42.	Oil Runner	
	9.	Impeller		31.	" Bushing	Upper	43	Flexible Coupling	Driving Half
1	0.	Stuff Box Bushing	Outer	32.	" "	Lower			
1	1.		Inner	33.	Oil Ring Fo	or Outer Bearing	44.		Driven "
1	2.	Distance Bushing		34.	" Cover "	" "	45.	Coupling Pins	
1	3.	Floating Clearance		35.	Sight Oil Gauge	Not Shown	46.	" Bushings	
		Ring							
1	4.	Plate.							
1	5.	"							
1	6.	Floating Clearance							
		Ring							
		Water Seal	Outer						
	8.		Inner	10	46				
		Gland	Outer	40	45 24 25 28	(J)	1	3130 34	38 38 40
			**	1	7/1/7/17	/		0 3/3	9/7/
		Gland	Inner		The state of			harman /	The state of the s
-	22.	" Bushing	"						
		Bearing Bracket	"						
	24.	" Cap							
		Bearing Bushing	Upper						W THE
	26.	" Bushing	Lower		26 //25			7 7 33	7//
		Oil Ring For	Inner Bearing		23 26 35	7		29 35	37 3942
	28.	" Cover "			27			/	
		Bearing Bracket	Outer			1	-0		
	30.	" Cap					-		
	31.	" Bushing	Upper Lower						
	32.	Oil Ring For							
	34.	" Cover "	" "			Fig.	1241		
	25	Sight Oil Gauge			at unn	ou p	0		
	эυ.	Signi on Gauge	TAGE SHOWII		Class "RD	S" Bearings and	Cou	plingSectional Vie	ew

Buffalo Class "RDS" Double Suction Multistage Centrifugal Pumps

DIMENSION TABLE



Dimensions D and E vary with size of Motor or Turbine being used. Certified Foundation Dimension Prints are furnished on all orders. Foundation Bolts are not furnished except on special order.

SPEED LIMITS OF BUFFALO CLASS "RDS" DOUBLE SUCTION MULTISTAGE PUMPS

Pump,	city.			250 1	bs. Max	imum W	Working Pressure						
of	Capacity s nute			Built in Two, Three and Four Stages									
Size of Inches	Normal Cap Gallons per Minute	Speed	80'	90'	100′	110'	120′	130′	140′	150′	160′	170′	180′
2	100	Min. Max.	1450 3400	1575 3550	1675 3700	1775 3850	1850 4000	1925 4000	2000 4000	2075 4000	2125 4000	2160 4000	2200 4000
2 1/2	155	Min. Max.	$\frac{1600}{3700}$	1735 3900	1840 4000	1950 4000	2035 4000	2115 4000	2200 4000	2280 4000	2385 4000	$\frac{2375}{4000}$	2420 4000
3	225	Min. Max.	$\frac{1360}{3200}$	$\frac{1480}{3365}$	$\frac{1570}{3495}$	$\frac{1660}{3625}$	1740 3750	1810 3750	1880 3750	1940 3750	1990 3750	2020 3750	2035 3750
4	400	Min. Max.	$^{1250}_{2525}$	$\frac{1370}{2680}$	$\frac{1455}{2785}$	$\frac{1540}{2885}$	1615 3000	1680 3000	1740 3000	1800 3000	1850 3000	1870 3000	1885 3000
5A	620	Min. Max.	$\frac{1325}{2650}$	$\frac{1450}{2800}$	$\frac{1525}{2900}$	$\frac{1625}{3000}$	1700 3000	1775 3000	1825 3000	1900 3000	1950 3000	1975 3000	2000 3000
5B	620	Min. Max.	$\frac{1025}{2050}$	$\frac{1125}{2150}$	$\frac{1200}{2275}$	$\frac{1250}{2400}$	$\frac{1300}{2500}$	1350 2600	1400 2600	1475 2600	1500 2600	1525 2600	1550 2600
6	900	Min. Max.	$\frac{1070}{2150}$	$\frac{1175}{2275}$	$\frac{1250}{2390}$	$\frac{1325}{2500}$	$\frac{1385}{2600}$	1440 2600	1490 2600	1540 2600	1575 2600	1600 2600	1625 2600
8	1600	Min. Max.	1000 1800	1090 1900	$\frac{1170}{2000}$	$\frac{1245}{2100}$	$\frac{1300}{2200}$	$\frac{1345}{2200}$	1390 2200	$\frac{1430}{2200}$	$\frac{1470}{2200}$	$\frac{1500}{2200}$	1525 2200

Note! These speeds are for one stage; for heads higher than shown on this table, divide the total head equally into two or more stages as required and apply speed per stage as above

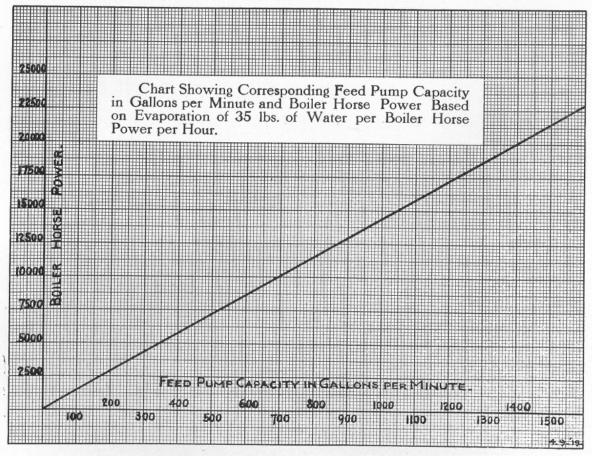


Fig. 1214
Pump Capacities for Various Boiler Horse Power

If Boiler Horse Power at Peak Loads exceeds boiler rating, pump capacity must be increased accordingly. Page 10

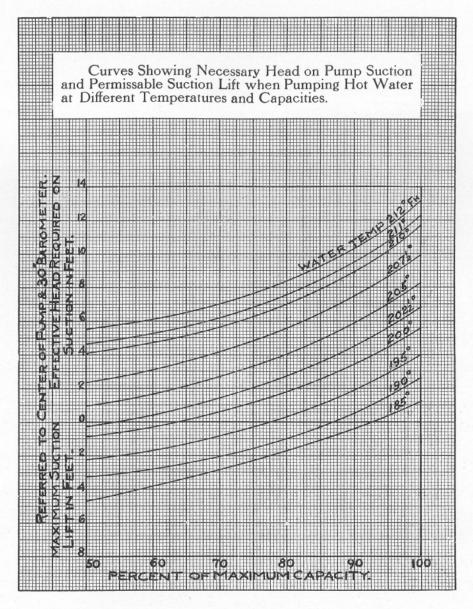


Fig. 1220 Head on Suction Necessary When Handling Hot Water

In all cases of boiler feeding or of pumping hot water for any service, water must flow to the pump suction under a positive head, sufficient to overcome the suction pull and pipe friction, and also to prevent vapor from forming in the pump casing.

It has been determined from experiments and from actual operation that a definite relationship exists between the temperature of the water, the effective head on the pump suction, and the capacity of the pump. The chart, Fig. 1220, shows the effective head necessary to obtain pump capacities from 50% to 100% of the maximum cold water capacity of the pump. Maximum cold water capacity ratings are given on page 7.

Effective head should be measured above the center line of the pump. Pipe connection between the pump suction and heater should be of ample size and as straight and direct as possible, so as to avoid high velocity and consequent pipe friction.

The "BUFFALO" line includes

STEAM PUMPS VACUUM PUMPS CONDENSERS POWER PUMPS CENTRIFUGAL PUMPS

"BUFFALO" pumps are used extensively for

Acid Plants Bilge and Drainage Boiler Feeding Chemical Plants

General Water Supply

Heating Systems Irrigation Projects Marine Service Mine Drainage Pulp and Paper Mills

Reclamation Projects Sewage Disposal

SCANNED BY: AEM OF LOCKPORT NY USA

POSTED ON: **SEPTEMBER 27, 2016**

EDITED BY: BRIAN D. SZAFRANSKI

ELMA, NEW YORK USA **COURTESY OF:**

WESTERN NY GAS & STEAM ENGINE ASSOCIATION

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NOTE: ORIGINAL DOCUMENT HAD WATER DAMAGE

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